

A pin in a knuckle joint is shown in part (a) of the figure. The joint is subject to a repeatedly applied and released load of 6000 N in tension. Assume the loading on the pin is modeled as concentrated forces as shown in part (b) of the figure. The shaft is made from AISI 1018 hot-rolled steel that has been machined to its final diameter. Based on a stress element on the outer surface at the cross-section A, determine a suitable diameter of the pin, rounded up to the next mm increment, to provide at least a factor of safety of 1.5 for both infinite fatigue life and for yielding. Table A-18

Table A-20 $S_y = 220 \text{ MPa}$
 $S_u = 400 \text{ MPa}$

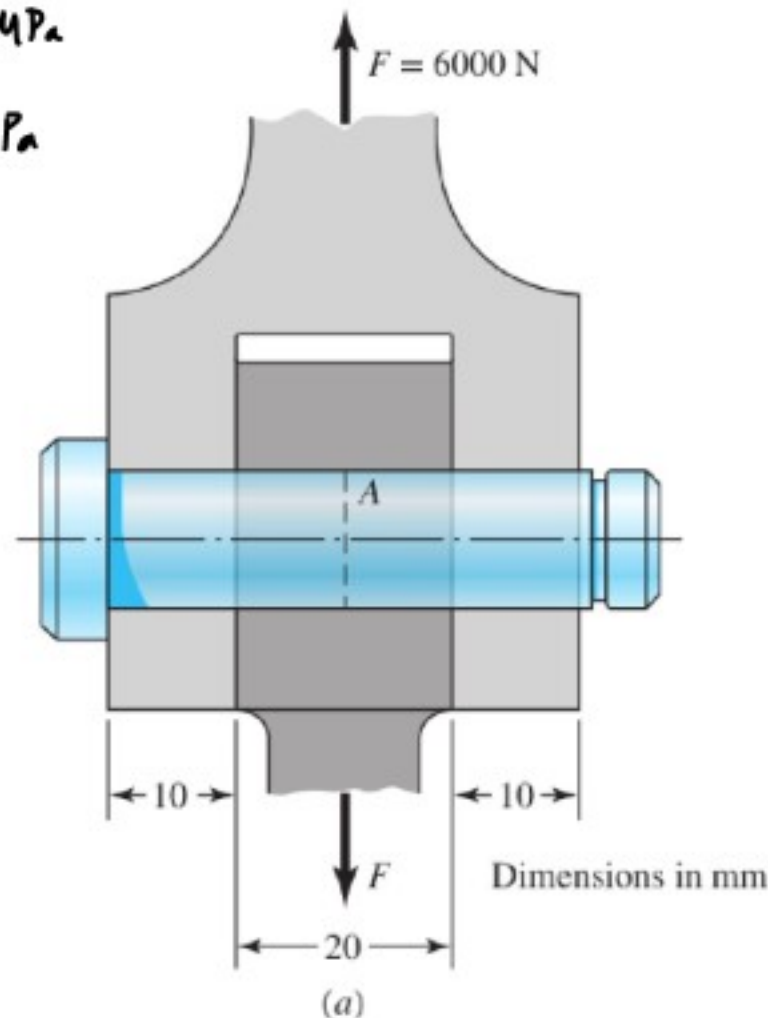
Table A-9: 5

$S_e' = 0.5(400) = 200 \text{ MPa}$

$S_e = S_e' K_a K_b K_c K_d K_e$ Sec 6-9
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 $= 200 \cdot 0.8 \cdot 1 \cdot 1 \cdot 1 \cdot 1 = 160 \text{ MPa}$

$d = 17 \text{ mm}$

$K_b = 0.92$



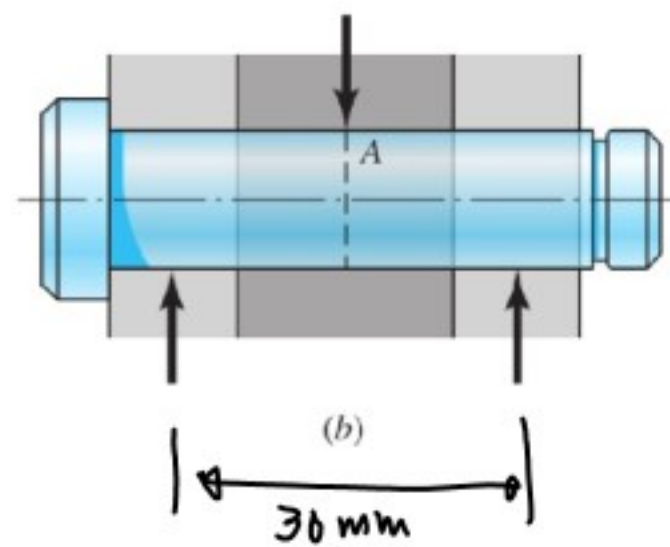
$M = \frac{F l}{4} = \frac{6000(0.03)}{4} = 45 \text{ N-m}$

$I = \frac{\pi d^4}{64}$

$\sigma = \frac{M c}{I} = \frac{45 \frac{d}{2}}{\frac{\pi d^4}{64}} = \frac{458}{d^3}$

$\frac{S_y}{\sigma} = 1.5$
 $\frac{220 \times 10^6}{1.5} = \sigma = \frac{458}{d^3}$

$d = 15 \text{ mm}$



A 1-in-diameter solid round bar has a groove 0.1-in deep with a 0.1-in radius machined into it. The bar is made of AISI 1020 CD steel and is subjected to a purely reversing torque of 1800 lbf · in.

(a) Estimate the number of cycles to failure.

(b) If the bar is also placed in an environment with a temperature of 750°F, estimate the number of cycles to failure.

Table A-20 $S_{ut} = 68 \text{ kpsi}$

Fig A-15-15

$D = 1 \text{ in}$ $D/d = 1.25$

$d = 0.8 \text{ in}$ $r/d = 0.125$
 $r = 0.1 \text{ in}$

$K_{ts} = 1.4$

Table A-13
 $\tau_{max} = \frac{T_c}{J} = \frac{1800 (0.4)}{\frac{\pi (0.8)^4}{32}} = 17.9 \text{ kpsi}$

$\sigma_{max} = \tau_{max}$

$\sigma_{min} = -\tau_{max}$

$\sigma_a = \left| \frac{\sigma_{max} - \sigma_{min}}{2} \right| = 17.9 \text{ kpsi}$

$S'_e = 0.5 S_{ut}$

$S_e = S'_e K_a K_b K_c K_d K_e = S'_e 0.79 \cdot 0.879 \cdot 0.59$

$J = \frac{\pi d^4}{32}$
 $= \frac{\pi (0.8)^4}{32}$

$N = \left(\frac{\sigma_a}{a} \right)^{1/b}$

$a = \frac{(\tau S_{ut})^2}{S_e}$

$b = -\frac{1}{3} \log \left(\frac{\tau S_{ut}}{S_e} \right)$

Fig 6-23
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