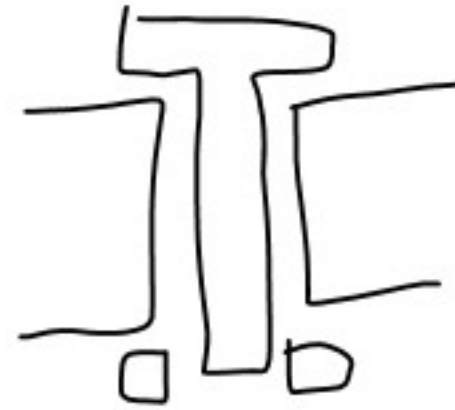


Tension Joints

P tension load per bolt

$$P = \frac{P_{total}}{N}$$



$$\frac{P_b}{k_b} = \Delta \delta_b = \Delta \delta_m = \frac{P_m}{k_m}$$

$$P_m = \frac{k_m}{k_b} P_b$$

$$P_b = \frac{k_b P}{k_b + k_m} = cP$$

$$P_m = P - P_b = (1 - c)P$$

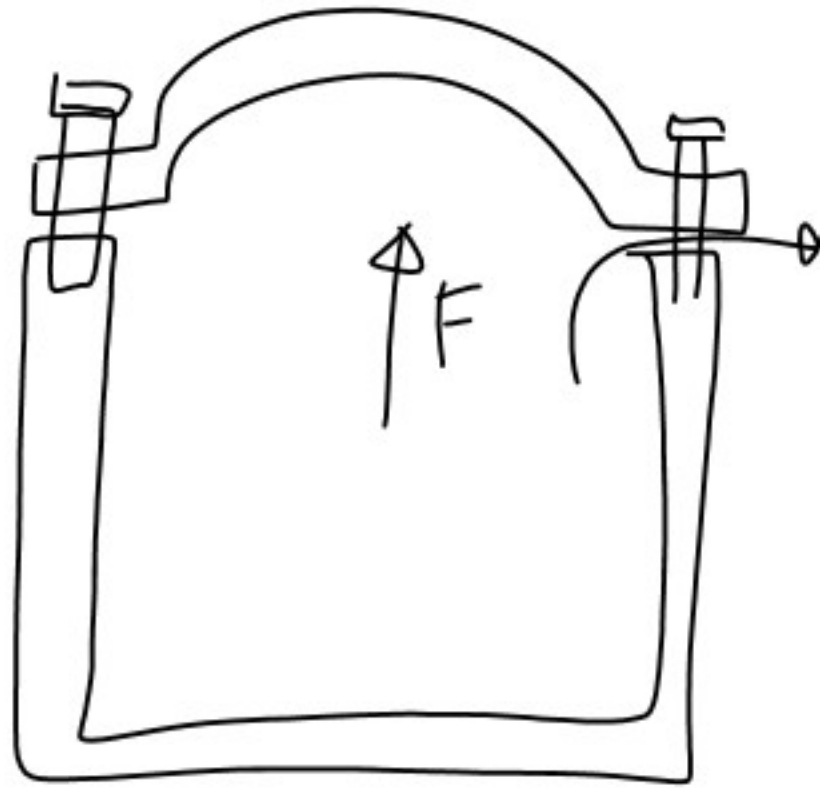
Stiffness
constant

$$c = \frac{k_b}{k_b + k_m}$$

F_i preload

$$F_b = P_b + F_i = cP + \bar{F}_i$$

$$F_m = P_m - \bar{F}_i = (1-c)P - F_i$$



Torque to preload

$$T = \frac{F_i d_m}{2} \left(\frac{l + \pi f d_m \sec \alpha}{\pi d_m - f l \sec \alpha} \right) + \frac{F_i f_c d_c}{2}$$

$$T = K F_i d$$

$$K = \left(\frac{d_m}{2d} \right) \left(\frac{\tan \lambda + f \sec \alpha}{1 - f \tan \lambda \sec \alpha} \right) + 0.625 f_c$$

Table 8-15 Page 450

Static Loading

$$\sigma_b = \frac{F_b}{A_t} = \frac{CP - F_i}{A_t}$$

$$n_p = \frac{S_p}{\sigma_b} = \frac{S_p A_t}{CP + F_i}$$

Load Factor

$$S_p = \frac{C_{UL} P + F_i}{A_t}$$

$$n_L = \frac{S_p A_t - F_i}{CP}$$

Factor of safety on Separation

$$n_o = \frac{F_i}{P(1-C)}$$

$$F_i = \begin{cases} 0.75 F_p & \text{reused fasteners} \\ 0.95 F_p & \text{permanent bolts} \end{cases}$$

$$F_p = A_t S_p \quad \text{proof force}$$

8-11 An M14 \times 2 hex-head bolt with a nut is used to clamp together two 15-mm steel plates.

- (a) Determine a suitable length for the bolt, rounded up to the nearest 5 mm.
- (b) Determine the bolt stiffness.
- (c) Determine the stiffness of the members.



$$L > l + H$$

$$L > 30 + 12.8$$

$$L > 42.8$$

$$L = 45 \text{ mm}$$

$$l = 15 \text{ mm} \cdot 2 = 30 \text{ mm}$$

$$H = 12.8 \text{ mm} \quad A-31$$

$$K_b = \frac{A_d A_t E}{A_d l_t + A_t l_d} = \frac{159 \text{ mm}^2 \cdot 115 \text{ mm}^2 \cdot 207 \text{ GPa}}{159 \text{ mm}^2 \cdot 19 \text{ mm} + 115 \text{ mm}^2 \cdot 11 \text{ mm}} = \frac{3665170 \text{ mm}^4 \text{ GPa}}{4191 \text{ mm}^3}$$

$$A_d = \pi \frac{d^2}{4} = \pi \frac{19^2}{4} = 159 \text{ mm}^2$$

$$A_t = 115 \text{ mm}^2 \quad \text{Table 8-1}$$

$$l_t = l - l_d = 30 - 11 = 19 \text{ mm}$$

$$l_d = L - L_T = 45 - (2d + 6) = 45 - (2 \cdot 19 + 6) = 11 \text{ mm}$$

$$E = 207 \text{ GPa}$$

$$= 875 \text{ mm GPa} \left(\frac{1 \text{ m}}{1000 \text{ mm}} \right) \left(\frac{1 \times 10^9 \text{ Pa}}{1 \text{ GPa}} \right) \left(\frac{1 \text{ N/m}^2}{1 \text{ Pa}} \right)$$

$$K_b = 875 \times 10^6 \frac{\text{N}}{\text{m}}$$

$$A = 0.78715 \quad B = 0.62873$$

$$\frac{K_m}{E_d} = A e^{B \frac{d}{\lambda}}$$

$$K_m = E_d A e^{B \frac{d}{\lambda}} \\ = 207 \text{ GPa} \cdot 19 \text{ mm} \cdot 0.78715 e^{\frac{0.62873 \cdot 19 \text{ mm}}{30 \text{ mm}}} = 3059 \text{ GPa mm} \left(\frac{1 \times 10^6 \text{ N/m}}{1 \text{ GPa mm}} \right) = \boxed{3.059 \times 10^9 \text{ N/m}}$$

8-29 For a bolted assembly with six bolts, the stiffness of each bolt is $k_b = 3$ Mlbf/in and the stiffness of the members is $k_m = 12$ Mlbf/in per bolt. An external load of 80 kips is applied to the entire joint. Assume the load is equally distributed to all the bolts. It has been determined to use $\frac{1}{2}$ in-13 UNC grade 8 bolts with rolled threads. Assume the bolts are preloaded to 75 percent of the proof load.

- Determine the yielding factor of safety.
- Determine the overload factor of safety.
- Determine the factor of safety based on joint separation.

$$P_{total} = 80 \text{ kips}$$

$$P = \frac{P_{total}}{N} = \frac{80}{6} = 13.33 \text{ kips}$$

$$F_i = 0.75 F_p \\ = 0.75 \cdot 17.03 = 12.77 \text{ kips}$$

$$S_p = 120 \text{ kpsi} \quad F_p = S_p A_t = 120 \cdot 0.1919 = 17.03 \text{ kips}$$

$$A_t = 0.1919 \text{ in}^2$$

$$S_y = 130 \text{ kpsi} \quad h = \frac{S_y A_t}{(cP + F_i)} = \frac{130 \text{ kpsi} \cdot 0.1919 \text{ in}^2}{0.2 \cdot 13.33 \text{ kips} + 12.77 \text{ kips}} = \boxed{1.2}$$

$$c = \frac{k_b}{k_b + k_m} = \frac{3}{3 + 12} = 0.2$$