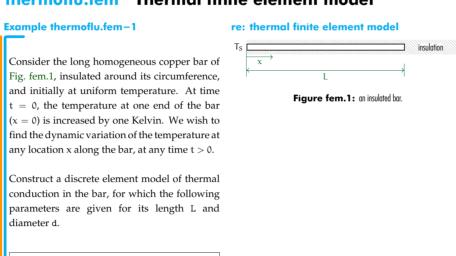
thermoflu.fem Thermal finite element model



L = 1; % m d = 0.01; % m

Geometrical considerations

The cross-sectional area for the bar is as follows.

a = pi/4*d^2; % m~2 x-sectional area

Dividing the bar into n sections ("finite elements") such that we have length of each dx

gives the following. n = 100; % number of chunks dx = L/n; % m ... length of chunk

Material considerations

The following are the material properties of copper.

cp = 390; % SI ... specific heat capacity rho = 8920; % SI ... density ks = 401; % SI ... thermal conductivity

Lumping

From the geometrical and material considerations above, we can develop a lumped thermal resistance R and thermal capacitance c of each cylindrical section of the bar of length dx. From Eq. 6 and Eq. 4, these parameters are as follows.

R = dx/(ks*a); % thermal resistance dV = dx*a; % m°3 ... section volume dm = rho*dV; % kg ... section mass c = dm*cp; % section weight

Linear graph model

The linear graph model is shown in Fig. fem.2 with the corresponding normal tree overlayed.

State-space model

The state variables are clearly the temperatures of C_i : T_{C_1}, \dots, T_{C_n} . Therefore, the order of the system is n.

The state, input, and output variables are $\mathbf{x} = \begin{bmatrix} \mathsf{T}_{\mathsf{C}_1} \cdots \mathsf{T}_{\mathsf{C}_n} \end{bmatrix}^{\top}, \quad \mathbf{u} = \begin{bmatrix} \mathsf{T}_{\mathsf{S}} \end{bmatrix}, \text{ and } \quad \mathbf{y} = \mathbf{x}.$

$$\mathbf{x} = \begin{bmatrix} \mathsf{T}_{\mathsf{C}_1} \cdots \mathsf{T}_{\mathsf{C}_n} \end{bmatrix}^\mathsf{T}, \quad \mathbf{u} = \begin{bmatrix} \mathsf{T}_{\mathsf{S}} \end{bmatrix}, \text{ and } \quad \mathbf{y} = \mathbf{y}$$

Elemental, continuity, and compatibility equations Consider the elemental, continuity, and compatibility equations, below, for the first, a middle, and the last elements. The following makes the assumption of homogeneity, which yields $R_i = R$ and $C_i = C$.

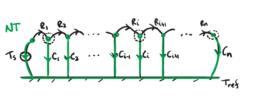


Figure fem.2: a linear graph of the insulated bar.

$ \dot{T}_{C_1} = \frac{1}{C}q_{C_1} \qquad q_{C_1} = q_{R_1} - q_{R_2} $ $ \dot{R}_1 \qquad q_{R_1} = \frac{1}{B}T_{R_1} \qquad \qquad T_{R_1} = T_S - T_{C_1} $	
R_1 $q_{R_1} = \frac{1}{R}T_{R_1}$ $T_{R_2} = T_S - T_{C_2}$	
. 101 Kul	
C_i $\dot{T}_{C_i} = \frac{1}{C} q_{C_i}$ $q_{C_i} = q_{R_i} - q_{R_{i+1}}$	
R_i $q_{R_i} = \frac{1}{R} T_{R_i}$ $T_{R_i} = T_{C_{i-1}} - T_{C_i}$	
C_n $\dot{T}_{C_n} = \frac{1}{C} q_{C_n}$ $q_{C_n} = q_{R_n}$	
$R_n q_{R_n} = \frac{1}{R} T_{R_n} T_{C_{n-1}} - T_{C_n}$	

Deriving the state equations for sections 1, i, and n For each of the first, a representative middle, and the last elements, we can derive the state equation with relatively few substitutions, as follows.

Let $\tau = RC$. The A and B matrices are, then



The outputs are the states: y = x. Or, in standard form with identity matrix I, the matrices are:

 $C = I_{n \times n}$ and $D = 0_{n \times 1}$. (3)

Simulation of a step response Define the A matrix.

A = zeros(n); % first row A(1,1) = -2/(R*c); A(1,2) = 1/(R*c);% last row A(n,n-1) = 1/(R*c); A(n,n) = -1/(R*c);% middle rows for i = 2:(n-1) A(i,i-1) = 1/(R*c); A(i,i) = -2/(R*c);A(i,i+1) = 1/(R*c);

Now define B, C, and D.

sys = ss(A,B,C,D);

y = step(sys,t);

B = zeros([n,1]); B(1) = 1/(R*c); C = eye(n); D = zeros([n,1]); Create a state-space model.

Simulate a unit step in the input temperature.

Tmax = 1200; % sec ... final sim time
t = linspace(0,Tmax,100);

Plot the step response To prepare for creating a 3D plot, we need to make a grid of points. x = dx/2:dx:(L-dx/2);[X,T] = meshgrid(x,t);

1200 5= 20 m.4

Now we're ready to plot. The result is shown in Fig. fem.3.

figure contourf(X,T,y) shading(gca,'interp') xlabel('x') ylabel('time') zlabel('temp (K)')

