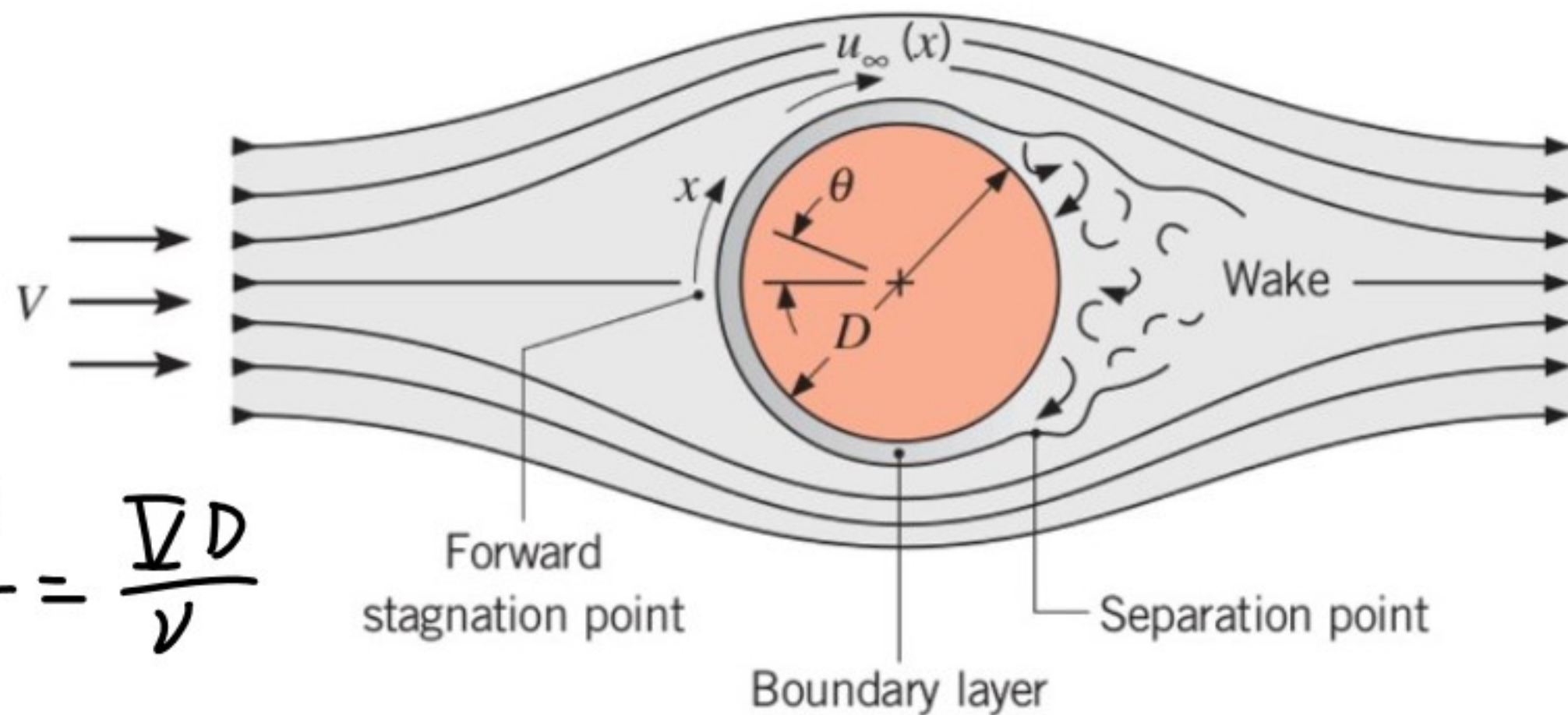
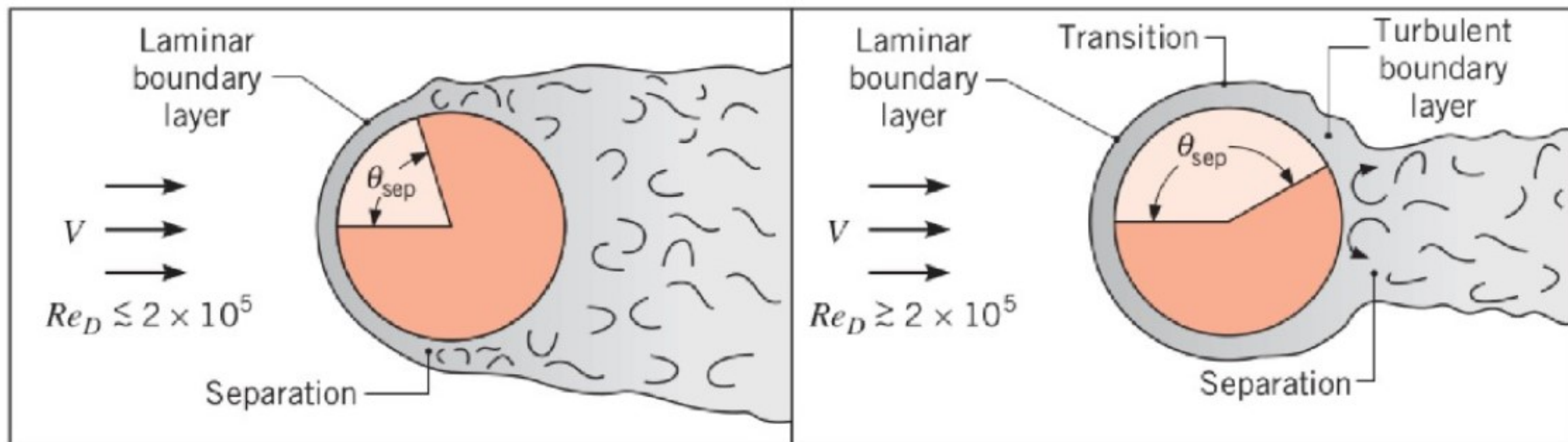


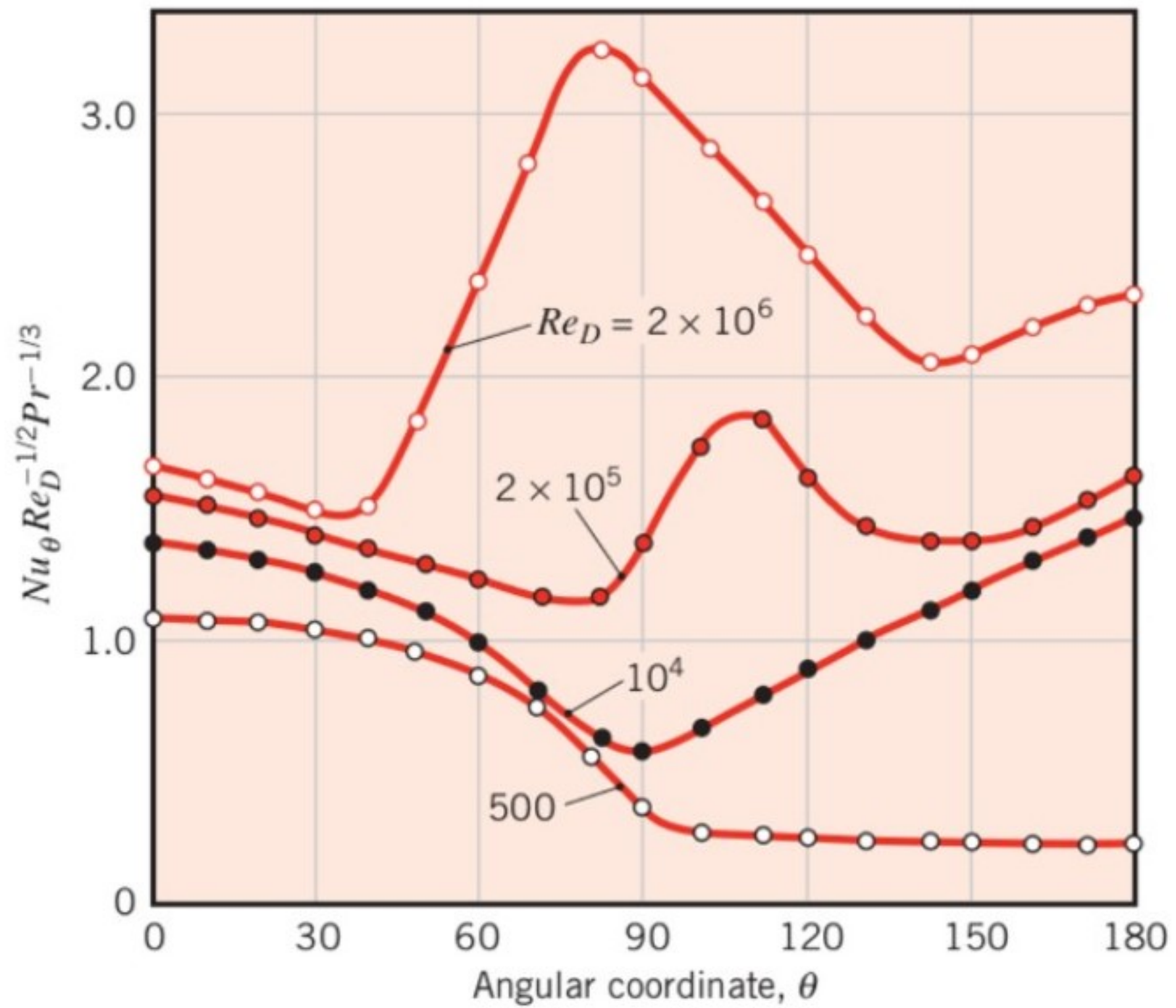
Cylinder in cross flow



$$Re_D = \frac{\rho V D}{\mu} = \frac{V D}{\nu}$$

V fluid velocity





At low Reynolds Numbers

$$Nu_D(\theta=0) = 1.15 Re_D^{1/2} Pr^{1/3}$$

For any Reynolds Number







$$\overline{Nu}_D = \frac{\bar{h} D}{k} = C Re_D^m Pr^{1/3} \quad Pr > 0.7$$

Table 7.2 and 7.3

TABLE 7.2 Constants of Equation 7.52 for
the circular cylinder in cross flow [12, 13]

Re_D	C	m
0.4–4	0.989	0.330
4–40	0.911	0.385
40–4000	0.683	0.466
4000–40,000	0.193	0.618
40,000–400,000	0.027	0.805

TABLE 7.3 Constants of Equation 7.52 for noncircular cylinders in cross flow of a gas [14, 15]^a

Geometry	Re_D	C	m
Square			
$V \rightarrow$  $\begin{matrix} \uparrow D \\ \downarrow \end{matrix}$	6000–60,000	0.304	0.59
$V \rightarrow$  $\begin{matrix} \uparrow D \\ \downarrow \end{matrix}$	5000–60,000	0.158	0.66
Hexagon			
$V \rightarrow$  $\begin{matrix} \uparrow D \\ \downarrow \end{matrix}$	5200–20,400	0.164	0.638
$V \rightarrow$  $\begin{matrix} \uparrow D \\ \downarrow \end{matrix}$	20,400–105,000	0.039	0.78
Thin plate perpendicular to flow			
$V \rightarrow$  $\begin{matrix} \uparrow D \\ \downarrow \end{matrix}$ Front	10,000–50,000	0.667	0.500
$V \rightarrow$  $\begin{matrix} \uparrow D \\ \downarrow \end{matrix}$ Back	7000–80,000	0.191	0.667

^aThese tabular values are based on the recommendations of Sparrow et al. [15] for air, with extension to other fluids through the $Pr^{1/3}$ dependence of Equation 7.52. A Prandtl number of $Pr = 0.7$ was assumed for the experimental results for air that are described in [15].

$$\overline{Nu}_D = C Re_D^m Pr^n \left(\frac{Pr}{Pr_s} \right)^{1/4}$$

$$0.7 < Pr < 500$$
$$1 < Re_D < 10^6$$

All properties at T_∞
except Pr_s at T_s

Table 7.9

if	$Pr < 10$	$n = 0.37$
if	$Pr > 10$	$n = 0.36$

TABLE 7.4 Constants of
Equation 7.53 for the circular
cylinder in cross flow [18]

Re_D	C	m
1–40	0.75	0.4
40–1000	0.51	0.5
10^3 – 2×10^5	0.26	0.6
2×10^5 – 10^6	0.076	0.7

$$\overline{Nu_D} = 0.3 + \frac{0.62 Re_D^{1/2} Pr^{1/3}}{(1 + (0.4/Pr)^{2/3})^{1/4}} \left(1 + \left(\frac{Re_D}{282000} \right)^{5/8} \right)^{4/5}$$

$$Re_D Pr > 0.2$$

7.35 An $L = 1$ -m-long vertical copper tube of inner diameter $D_i = 20$ mm and wall thickness $t = 2$ mm contains liquid water at $T_w \approx 0^\circ\text{C}$. On a winter day, air at $V = 3$ m/s, $T_\infty = -20^\circ\text{C}$ is in cross flow over the tube.

- (a) Determine the rate of heat loss per unit mass from the water (W/kg) when the tube is full of water.
- (b) Determine the rate of heat loss from the water (W/kg) when the tube is half full.

$$Re_D = \frac{VD}{\nu} = \frac{3 \text{ m/s} \cdot 0.024 \text{ m}}{11.4 \times 10^{-6} \text{ m}^2/\text{s}}$$

$$= 6316$$

$$\overline{Nu_D} = C Re_D^m Pr^{1/3} = 0.193 (6316)^{0.698} (0.72)^{1/3} = \frac{\bar{h} D}{k}$$