

ss.sfdbck Controller design method

We will consider single-input single-output (SISO) control plants that can be written with input u ; state vector x ; output y ; state model matrices A , B , C , and D ; and state and output equations

$$\dot{x} = Ax + Bu \tag{1a}$$

$$y = Cx + Du. \tag{1b}$$

Plants of this form can be written in block diagram form, as illustrated in Fig. sfdbck.1. In general, SISO systems are of order n with n state variables.

Let us consider the following feedback control method called state feedback control. We will feed back the state vector x , operate on it with a $1 \times n$ vector of gains $K \in \mathbb{R}^n$, and subtract the result from the command r , the result of which becomes the input u , as shown in Fig. sfdbck.2. The control problem for state feedback control is to determine the n gains in K such that the closed-loop poles are located in desirable positions. The gain $N \in \mathbb{R}$ is provided for steady-state error considerations, which will be addressed in Lec. ss.sfdbck. A new state model can be derived for the closed-loop system as follows. Let us consider the command r to be our new “input,” instead of u , which is now the control effort. From the block diagram,

$$u = Nr - Kx, \tag{2}$$

which can be substituted into Eq. 1 to define the new state model

$$\dot{x} = (A - BK)x + NBr \tag{3a}$$

$$y = (C - DK)x + NDr. \tag{3b}$$

The eigenvalues of $A - BK$, which can be found from equating zero and the closed-loop characteristic polynomial

$$P_K = \det (sI - A + BK), \tag{4}$$

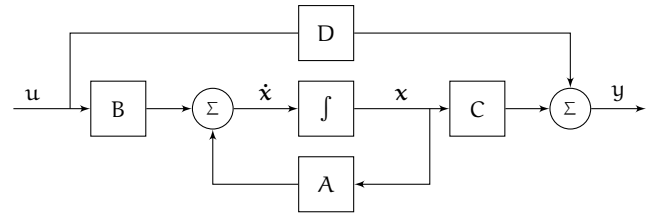


Figure sfdbck.1: the plant state model of Eq. 1 written in block diagram form.

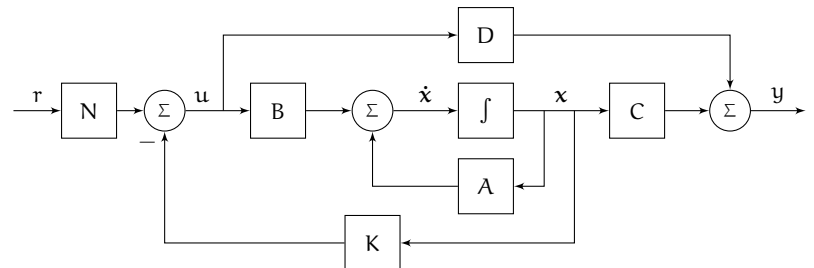


Figure sfdbck.2: the state feedback control block diagram.

are equal to the closed-loop poles, which we would like to place in specific locations. Those specific locations can be specified by the design characteristic polynomial P_d . P_K depends on the n gains K_i , and n equations can be found by equating the polynomial coefficients of P_K and P_d .

Solving for K_i is straightforward but can be very tedious in the general case. Let the coefficients of P_d be δ_i and those of P_K be denoted κ_i . Then the $n \times 1$ vector containing κ_i can be expressed as a linear combination of K_i as

$$\kappa = \mathcal{K} \mathbf{K}^T, \quad (5)$$

where \mathcal{K} is an $n \times n$ matrix of coefficients that were derived from A and B . Let δ be the $n \times 1$ vector of components δ_i . Since the vector δ is specified by our design requirements, we can solve for \mathbf{K} as follows.

$$\kappa = \delta, \quad (6)$$

and therefore,

$$\begin{aligned} \mathcal{K} \mathbf{K}^T = \delta &\implies \\ \mathbf{K}^T = \mathcal{K}^{-1} \delta &\implies \\ \mathbf{K} = (\mathcal{K}^{-1} \delta)^T. &\quad (7) \end{aligned}$$

Eq. 7 is valid for all cases in which \mathcal{K} is invertible.¹ However, there is a special form of the original state-space model that always yields a simple solution for \mathbf{K} : the phase-variable canonical form (see [Appendix B.02](#)).

1. We leave the following as an open question: under what conditions is \mathcal{K} invertible?

Solving for the gain via the phase-variable canonical form

The phase-variable canonical form of the original system is:

$$\dot{\mathbf{x}}_c = \mathbf{A}_c \mathbf{x}_c + \mathbf{B}_c \mathbf{u} \quad (8a)$$

$$\mathbf{y} = \mathbf{C}_c \mathbf{x}_c + \mathbf{D}_c \mathbf{u} \quad (8b)$$

where

$$A_c = \begin{bmatrix} 0 & 1 & 0 & \cdots & 0 & 0 \\ 0 & 0 & 1 & & 0 & 0 \\ \vdots & \vdots & & \ddots & & \vdots \\ 0 & 0 & 0 & & 1 & 0 \\ 0 & 0 & 0 & \cdots & 0 & 1 \\ -a_0 & -a_1 & -a_2 & \cdots & -a_{n-2} & -a_{n-1} \end{bmatrix}, \quad B_c = \begin{bmatrix} 0 \\ 0 \\ \vdots \\ 0 \\ 0 \\ 1 \end{bmatrix}, \tag{8c}$$

$$C_c = [c_1 \quad c_2 \quad \cdots \quad c_n], \text{ and} \quad D_c = [d_1], \tag{8d}$$

where the components a_i are defined by the original characteristic polynomial

$$P = \det(sI - A) = s^n + a_{n-1}s^{n-1} + \cdots + a_1s + a_0. \tag{9}$$

With A_c defined, the form of the feedback state model with feedback row vector K_c is:

$$A'_c = A_c - B_c K_c, \quad B'_c = B_c, \tag{10a}$$

$$C'_c = C_c - D_c K_c, \text{ and} \quad D'_c = D_c. \tag{10b}$$

A'_c deserves further attention. The special canonical form of A_c and B_c makes the expression for A'_c simply

$$A'_c = \begin{bmatrix} 0 & 1 & \cdots & 0 \\ 0 & 0 & & 0 \\ \vdots & \vdots & \ddots & \vdots \\ 0 & 0 & & 0 \\ 0 & 0 & \cdots & 1 \\ -(a_0 + K'_1) & -(a_1 + K'_2) & \cdots & -(a_{n-1} + K'_n) \end{bmatrix}, \tag{11}$$

where K'_i is the row vector of gains in the phase-variable canonical basis. The design characteristic polynomial coefficients δ_i must equal the characteristic polynomial coefficients

$$\delta_i = a_i + K'_{i+1}, \tag{12}$$

which gives

$$K'_i = \delta_{i-1} - a_{i-1}. \tag{13}$$

This yields K' . If we equate the feedback

$$\begin{aligned} Kx &= K'x_c \implies \\ K &= K'T_c. \end{aligned} \tag{14}$$

Let u and u_c be the controllability matrices for the original basis and the phase-variable canonical basis, respectively. From [Appendix B.02](#), we can compute the transformation matrix to be

$$T_c = u_c u^{-1}. \tag{15}$$

Steady-state error

We can use the gain N to drive the closed-loop steady-state error to zero for step inputs. The idea is that we can scale the input by the reciprocal of the closed-loop steady-state error. Let $G_{CL}(s)$ be the closed-loop transfer function. From the final value theorem for a unit step input,

$$N = \lim_{s \rightarrow 0, N \rightarrow 1} 1/G_{CL}(s). \tag{16}$$

If N is nonzero and finite, the response will have zero steady-state error. Although it is derived from unit step inputs, we can apply this formula to slowly varying inputs as well.

Example ss.sfdbck-1

re: state feedback pole placement design

Given the state-space model

$$\begin{aligned} A &= \begin{bmatrix} -1 & 0 & -1 \\ -1 & -1 & 0 \\ 0 & -1 & -1 \end{bmatrix} & B &= \begin{bmatrix} 1 \\ 0 \\ 0 \end{bmatrix} \\ C &= \begin{bmatrix} 0 & 0 & 1 \end{bmatrix} & D &= \begin{bmatrix} 0 \end{bmatrix}, \end{aligned}$$

- design a controller with 15% overshoot and a
- settling time of 1 sec.





